

# Computational Line Geometry as a Tool for Solving Engineering Problems

International Workshop on Line Geometry and Kinematics

Πάφος, 2011 - 04 -29

## **Geometry and Kinematics of Car Side Windows**

The Task

Obtaining the Optimal Screw Motion

Additional Constraints

Benefits

## **The Rope Ladder Problem**

Problem Formulation

The Planar Case

The Spatial Case

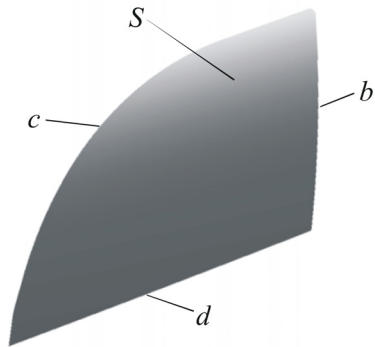
An Interpolation Problem

## Car Side Windows: The Task

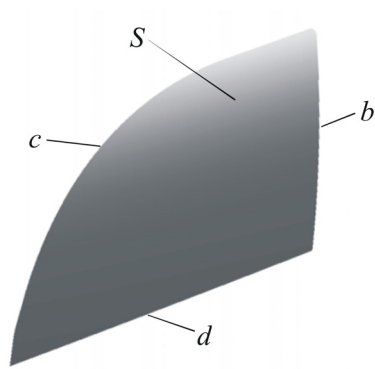
Geometry and Kinematics of Car Side Windows: The Task



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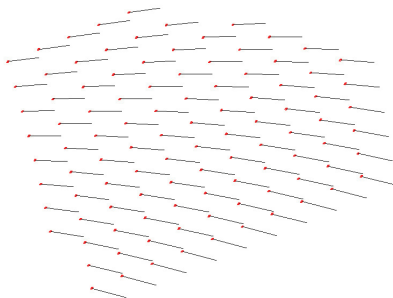
Geometry and Kinematics of Car Side Windows: The Task



- S* ... window surface
- d* ... daylight line
- b* ... b-pillar
- c* ... roof line

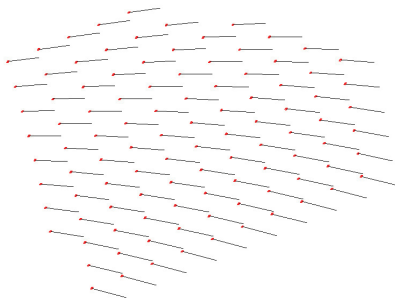
**Given:** A surface  $S$ ;  
(side window suggested by the stylist)

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Can  $S$  be moved in itself?

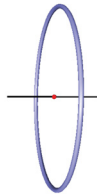


Which spatial curves can be moved **in themselves**?

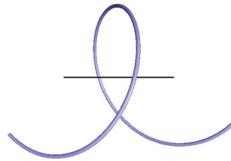
Which spatial curves can be moved **in themselves**?



straight line



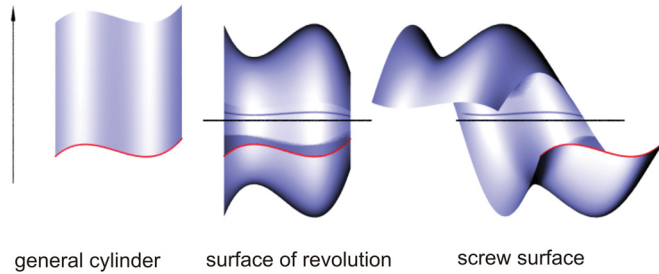
circle



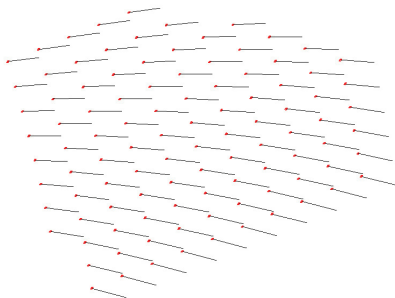
screw line

Which surfaces can be moved **in themselves**?

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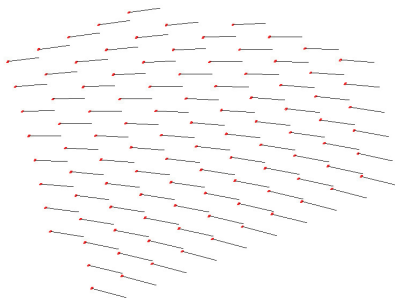


Geometry and Kinematics of Car Side Windows: The Task



Which screw motion best fits the surface  $S$ ?

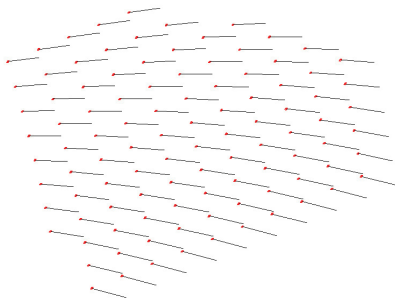
Geometry and Kinematics of Car Side Windows: The Task



Which screw motion best fits the surface  $S$ ?

axis  $a$ , screw parameter  $p$



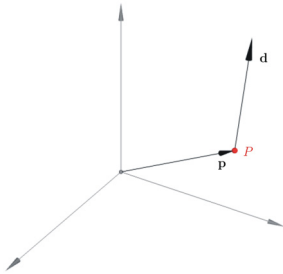


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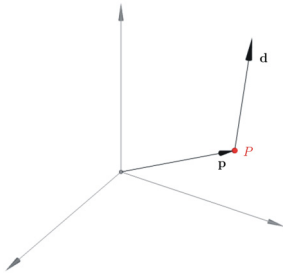
How to find this screw motion?

## Car Side Windows: Obtaining the Optimal Screw Motion



straight line  $g$ :

$$\mathbf{p} = \begin{bmatrix} p_{i1} \\ p_{i2} \\ p_{i3} \end{bmatrix}, \quad \mathbf{d} = \begin{bmatrix} d_{i1} \\ d_{i2} \\ d_{i3} \end{bmatrix}$$

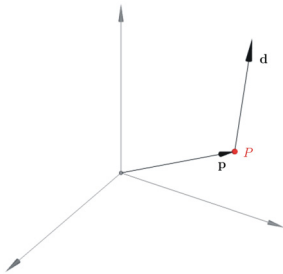


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$$\text{point } G: \begin{bmatrix} \mathbf{g} \\ \bar{\mathbf{g}} \end{bmatrix}, \quad \mathbf{g} = \mathbf{d}, \quad \bar{\mathbf{g}} = \mathbf{p} \times \mathbf{d}$$

### 3-dimensional space $\mathbb{E}^3$

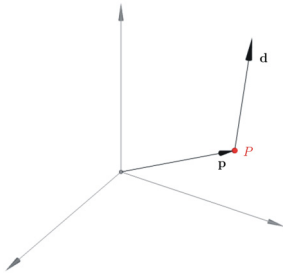


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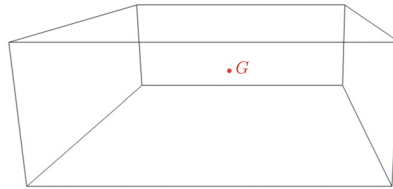
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straight line  $g$ :

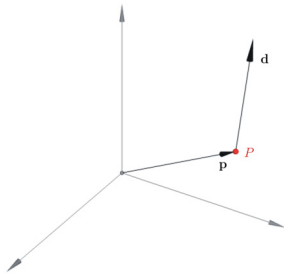
$$\mathbf{p} = \begin{bmatrix} p_{i1} \\ p_{i2} \\ p_{i3} \end{bmatrix}, \quad \mathbf{d} = \begin{bmatrix} d_{i1} \\ d_{i2} \\ d_{i3} \end{bmatrix}$$

### 5-dimensional space $\mathbb{P}^5$



$$\text{point } G: \begin{bmatrix} \mathbf{g} \\ \bar{\mathbf{g}} \end{bmatrix}, \quad \mathbf{g} = \mathbf{d}, \quad \bar{\mathbf{g}} = \mathbf{p} \times \mathbf{d}$$

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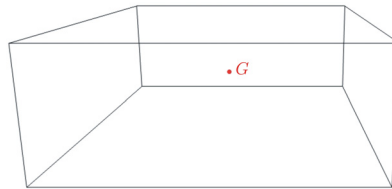
straight line  $g$ :

$$\mathbf{p} = \begin{bmatrix} p_{i1} \\ p_{i2} \\ p_{i3} \end{bmatrix}, \quad \mathbf{d} = \begin{bmatrix} d_{i1} \\ d_{i2} \\ d_{i3} \end{bmatrix}$$

Plücker condition:  $\langle \mathbf{g}, \bar{\mathbf{g}} \rangle = g_1 \bar{g}_1 + g_2 \bar{g}_2 + g_3 \bar{g}_3 = 0$

Klein quadric  $\mathcal{M}_2^4 \subset \mathbb{P}^5$

### 5-dimensional space $\mathbb{P}^5$



point  $G$ :  $\begin{bmatrix} \mathbf{g} \\ \bar{\mathbf{g}} \end{bmatrix}$ ,  $\mathbf{g} = \mathbf{d}$ ,  $\bar{\mathbf{g}} = \mathbf{p} \times \mathbf{d}$

Obtaining the Optimal Screw Motion

3-dimensional space  $\mathbb{E}^3$



screw motion  $M$

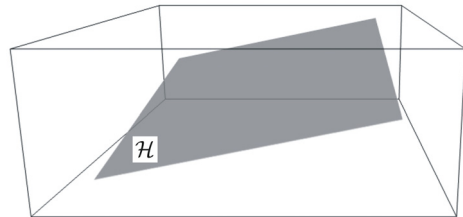


3-dimensional space  $\mathbb{E}^3$



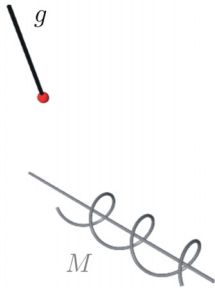
screw motion  $M$

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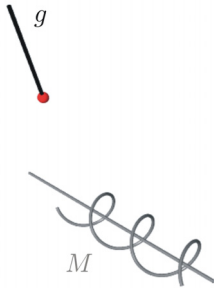
hyperplane  $\mathcal{H}$

3-dimensional space  $\mathbb{E}^3$



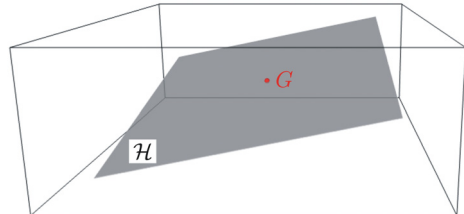
normal  $g$  belongs to the  
linear complex  $\mathcal{L}_M$

3-dimensional space  $\mathbb{E}^3$



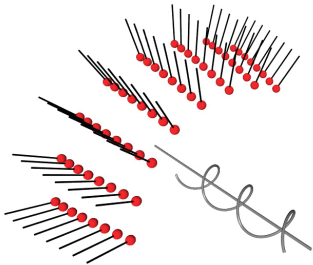
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5-dimensional space  $\mathbb{P}^5$

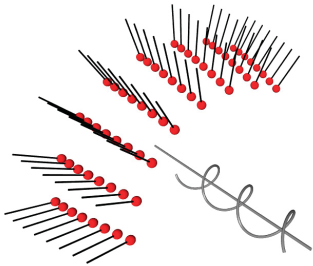


Point  $G$  lies in the hyperplane  $\mathcal{H}$

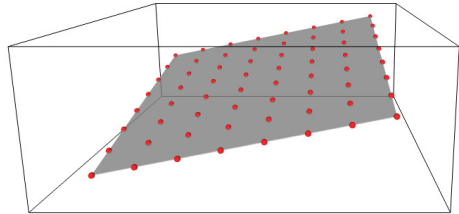
3-dimensional space  $\mathbb{E}^3$



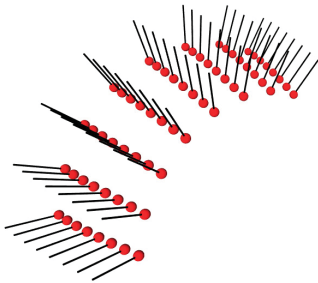
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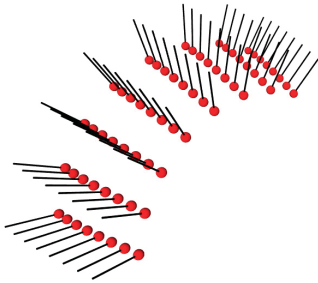
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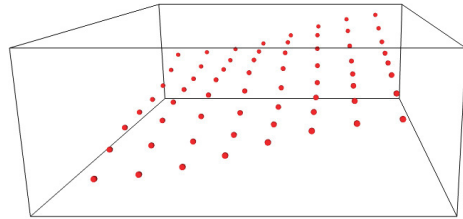
3-dimensional space  $\mathbb{E}^3$



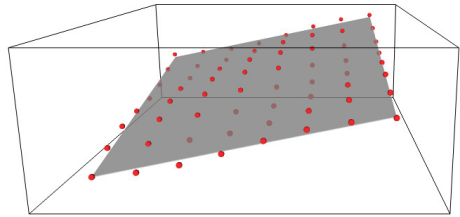
3-dimensional space  $\mathbb{P}^3$



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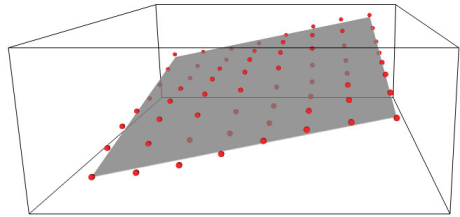


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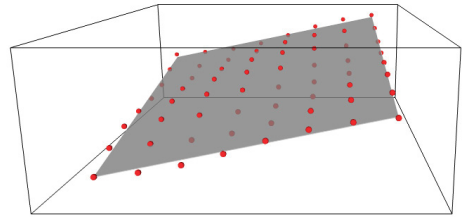
hyperplane of regression  $\mathcal{H}$

3-dimensional space  $\mathbb{P}^3$



yields the optimal screw motion

5-dimensional space  $\mathbb{P}^5$



hyperplane of regression  $\mathcal{H}$

$$\mathbf{v} \cdot \mathbf{g}_i + \mathbf{w} \cdot \overline{\mathbf{g}}_i = 0, \quad i = 1, \dots, n$$

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minimize the squared error function

$$e(\mathbf{v}, \mathbf{w}) := [\mathbf{v}^T, \mathbf{w}^T] \cdot \mathbf{S} \cdot \begin{bmatrix} \mathbf{v} \\ \mathbf{w} \end{bmatrix}$$

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minimize the squared error function

$$e(\mathbf{v}, \mathbf{w}) := [\mathbf{v}^\top, \mathbf{w}^\top] \cdot \mathbf{S} \cdot \begin{bmatrix} \mathbf{v} \\ \mathbf{w} \end{bmatrix}$$

where  $\mathbf{S}$  is the positive semidefinite  $6 \times 6$ -matrix "scatter matrix":

$$\mathbf{S} = \begin{bmatrix} \mathbf{g}_1 & \cdots & \mathbf{g}_n \\ \bar{\mathbf{g}}_1 & \cdots & \bar{\mathbf{g}}_n \end{bmatrix} \cdot \begin{bmatrix} \mathbf{g}_1^\top & \bar{\mathbf{g}}_1^\top \\ \vdots & \vdots \\ \mathbf{g}_n^\top & \bar{\mathbf{g}}_n^\top \end{bmatrix} = \begin{bmatrix} \sum_{i=1}^n \mathbf{g}_i \mathbf{g}_i^\top & \sum_{i=1}^n \mathbf{g}_i \bar{\mathbf{g}}_i^\top \\ \sum_{i=1}^n \bar{\mathbf{g}}_i \mathbf{g}_i^\top & \sum_{i=1}^n \bar{\mathbf{g}}_i \bar{\mathbf{g}}_i^\top \end{bmatrix}$$

$$\mathbf{v} \cdot \mathbf{g}_i + \mathbf{w} \cdot \bar{\mathbf{g}}_i = 0, \quad i = 1, \dots, n$$

minimize the squared error function

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subject to the normalizing constraint:

$$[\mathbf{v}^T, \mathbf{w}^T] \cdot \mathbf{E}_6 \cdot \begin{bmatrix} \mathbf{v} \\ \mathbf{w} \end{bmatrix} = w_1^2 + w_2^2 + w_3^2 = 1$$

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where  $\mathbf{E}_6 = \begin{bmatrix} \mathbf{I}_3 & \mathbf{O}_3 \\ \mathbf{O}_3 & \mathbf{O}_3 \end{bmatrix}$

Lagrangian multiplier method yields

$$(\mathbf{S} - \lambda \mathbf{E}_6) \cdot \begin{bmatrix} \mathbf{v} \\ \mathbf{w} \end{bmatrix} = \begin{bmatrix} 0 \\ \vdots \\ 0 \end{bmatrix}$$



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generalized eigenvalue problem

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determine the eigenvector space for the smallest eigenvalue  $\lambda_0$ :  
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screw parameter:  $p = \frac{\langle \mathbf{w}_0, \mathbf{v}_0 \rangle}{\langle \mathbf{w}_0, \mathbf{w}_0 \rangle}$

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direction vector of the screw axis  $a$ :  $\mathbf{d} = \mathbf{w}_0$

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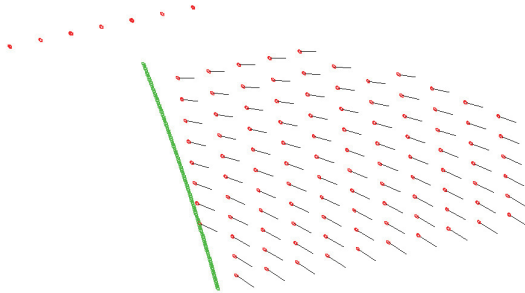
point on  $a$ :  $\mathbf{a} = \frac{\mathbf{w}_0 \times \mathbf{v}_0}{\langle \mathbf{w}_0, \mathbf{w}_0 \rangle}$

## Car Side Windows: Additional Constraints

Geometry and Kinematics of Car Side Windows: Additional Constraints



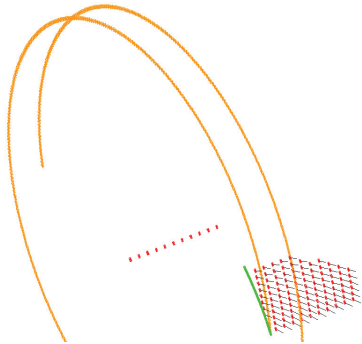
Geometry and Kinematics of Car Side Windows: Additional Constraints



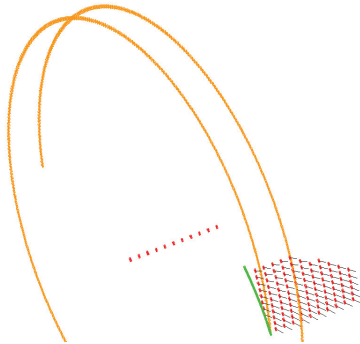
boundary curve  $b$  (b-pillar)



Geometry and Kinematics of Car Side Windows: Additional Constraints

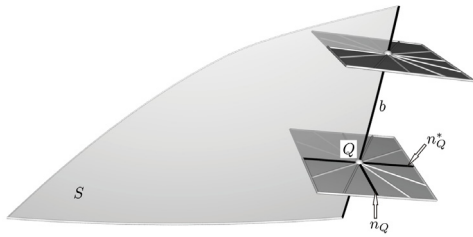


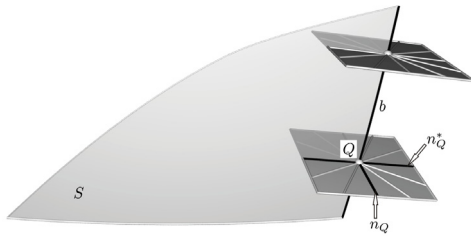
Geometry and Kinematics of Car Side Windows: Additional Constraints



$b$  is **not!** a trajectory of the optimal screw motion  $M$

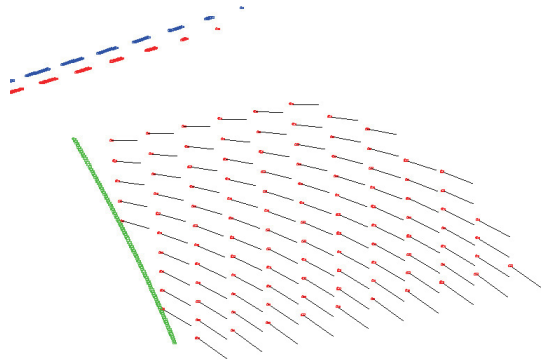
Geometry and Kinematics of Car Side Windows: Additional Constraints





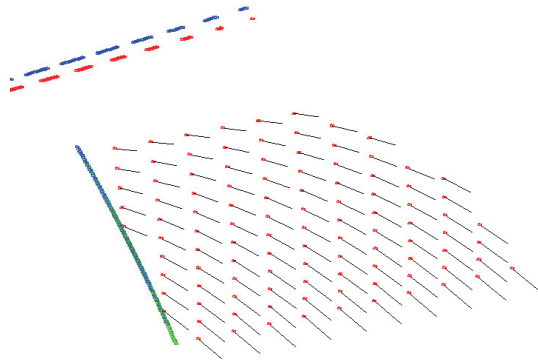
add some normals of the prescribed b-pillar  $b$  to the interpolation problem input

Geometry and Kinematics of Car Side Windows: Additional Constraints



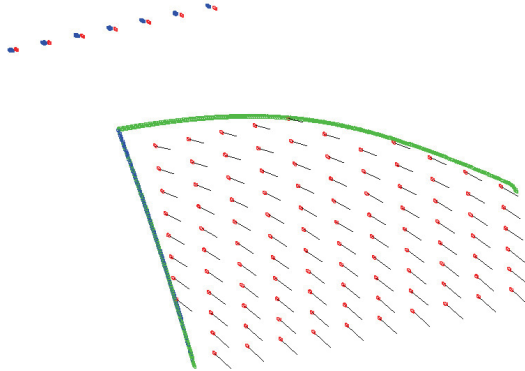
computing the optimal screw motion for both,  $S$  and  $b$

Geometry and Kinematics of Car Side Windows: Additional Constraints



delivers some replacement for the b-pillar  $b$

Geometry and Kinematics of Car Side Windows: Additional Constraints



the optimal screw motion  
delivers the optimal side window sheet (screw surface  $S$ )  
out of the roofline  $c$

## Car Side Windows: Benefits



## Car Side Windows: Benefits

engineering workload  
constructing the side window



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quality of the outcome

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serial production cost

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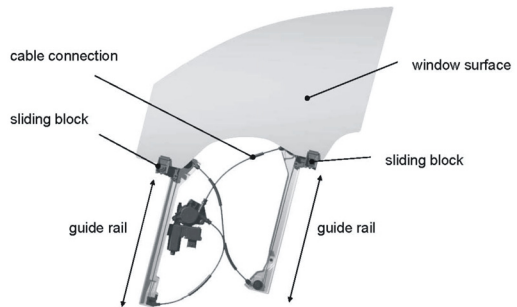


serial production cost



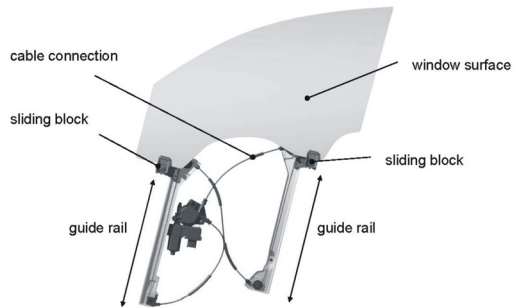
## Rope Ladder Problem: Problem Formulation

**Rope Ladder Problem:** Problem Formulation

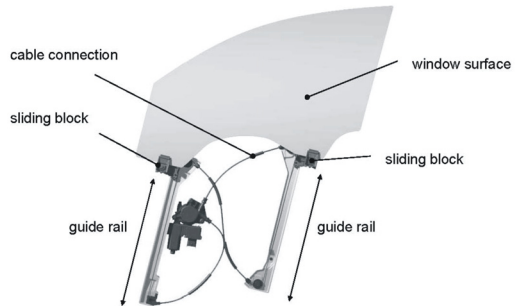
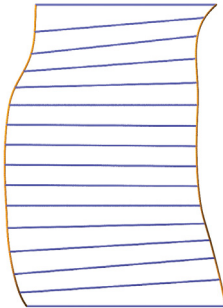




How can you move a rod so that its endpoint paths have equal length?



How can you move a rod so that its endpoint paths have equal length?



**Rope Ladder Problem:** Problem Formulation

$$A \dots \vec{OA} = \mathbf{a}(t), B \dots \vec{OB} = \mathbf{b}(t)$$

Rope Ladder Problem: Problem Formulation

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$$\text{dist}^2(A, B) = \langle \mathbf{b} - \mathbf{a}, \mathbf{b} - \mathbf{a} \rangle = d^2 = \text{const.}$$

**Rope Ladder Problem:** Problem Formulation

$$A \dots \vec{OA} = \mathbf{a}(t), B \dots \vec{OB} = \mathbf{b}(t)$$

$$\text{dist}^2(A, B) = \langle \mathbf{b} - \mathbf{a}, \mathbf{b} - \mathbf{a} \rangle = d^2 = \text{const.}$$

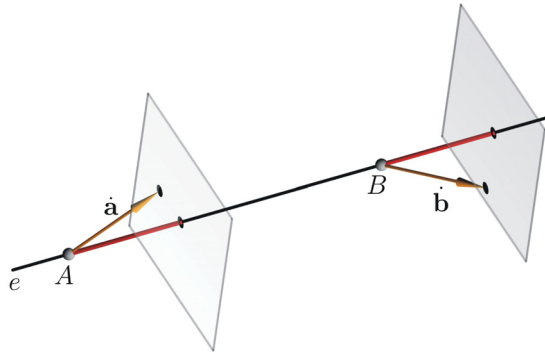
$$\langle \dot{\mathbf{a}}, \mathbf{b} - \mathbf{a} \rangle = \langle \dot{\mathbf{b}}, \mathbf{b} - \mathbf{a} \rangle, \text{ projection theorem}$$

Rope Ladder Problem: Problem Formulation

$$A \dots \vec{OA} = \mathbf{a}(t), B \dots \vec{OB} = \mathbf{b}(t)$$

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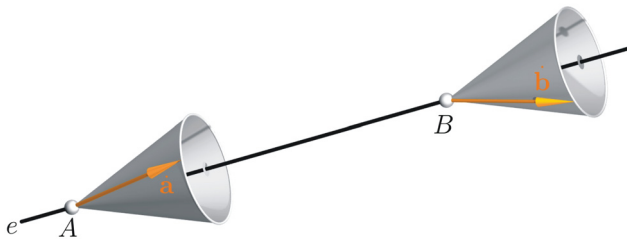
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$$|\dot{\mathbf{a}}| = |\dot{\mathbf{b}}| \implies \angle(\vec{AB}, \dot{\mathbf{a}}) = \angle(\vec{AB}, \dot{\mathbf{b}})$$



**Rope Ladder Problem:** Problem Formulation

$\Sigma$  ... moving system



**Rope Ladder Problem:** Problem Formulation

$\Sigma$  ... moving system

$\Sigma^*$  ... fixed system

**Rope Ladder Problem:** Problem Formulation

$\Sigma$  ... moving system

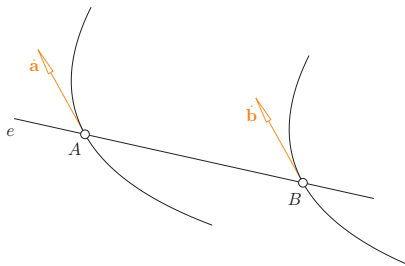
$\Sigma^*$  ... fixed system

$\Sigma/\Sigma^*$  ... motion

## Rope Ladder Problem: The Planar Case

planar case A

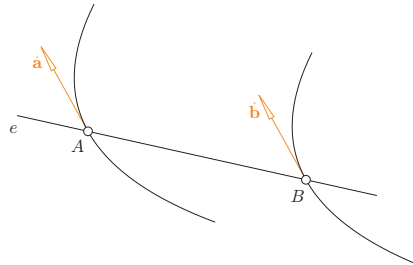
$$\angle(\vec{AB}, \vec{a}) = \angle(\vec{AB}, \vec{b}) \text{ for all } t \in [t_0, t_1]$$



planar case A

$$\angle(\vec{AB}, \dot{\mathbf{a}}) = \angle(\vec{AB}, \dot{\mathbf{b}}) \text{ for all } t \in [t_0, t_1]$$

$$\dot{\mathbf{a}} = \dot{\mathbf{b}} \text{ for all } t \in [t_0, t_1]$$

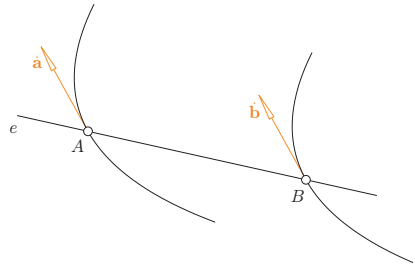


planar case A

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$$\dot{\mathbf{a}} = \dot{\mathbf{b}} \text{ for all } t \in [t_0, t_1]$$

curved translation

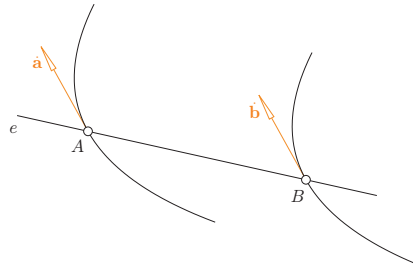


planar case A

$$\angle(\vec{AB}, \dot{\mathbf{a}}) = \angle(\vec{AB}, \dot{\mathbf{b}}) \text{ for all } t \in [t_0, t_1]$$

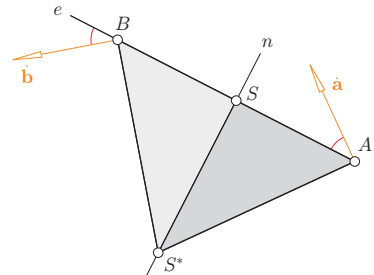
$$\dot{\mathbf{a}} = \dot{\mathbf{b}} \text{ for all } t \in [t_0, t_1]$$

curved translation



planar case B

$$\angle(\vec{AB}, \dot{\mathbf{a}}) = -\angle(\vec{AB}, \dot{\mathbf{b}}) \text{ for all } t \in [t_0, t_1]$$

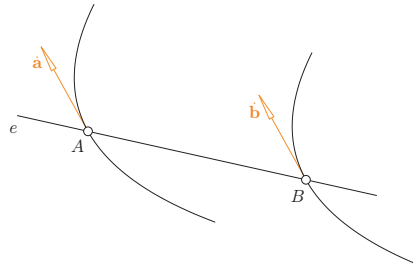


planar case A

$$\angle(\vec{AB}, \dot{\mathbf{a}}) = \angle(\vec{AB}, \dot{\mathbf{b}}) \text{ for all } t \in [t_0, t_1]$$

$$\dot{\mathbf{a}} = \dot{\mathbf{b}} \text{ for all } t \in [t_0, t_1]$$

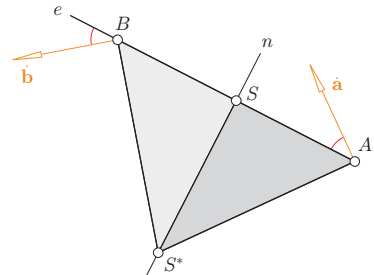
curved translation



planar case B

$$\angle(\vec{AB}, \dot{\mathbf{a}}) = -\angle(\vec{AB}, \dot{\mathbf{b}}) \text{ for all } t \in [t_0, t_1]$$

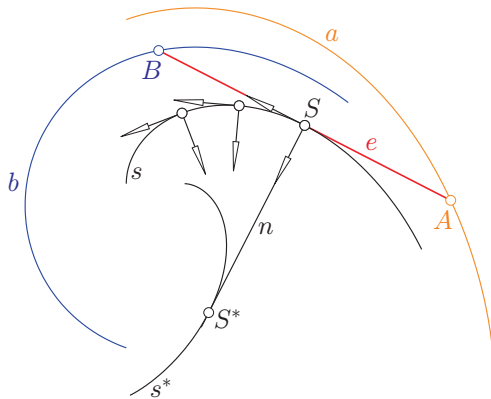
bisector  $n$  of  $AB =$  moving polhode





planar case B

$\Sigma/\Sigma^*$  ... motion of a straight line  $n$  rolling on a curve  $s^*$

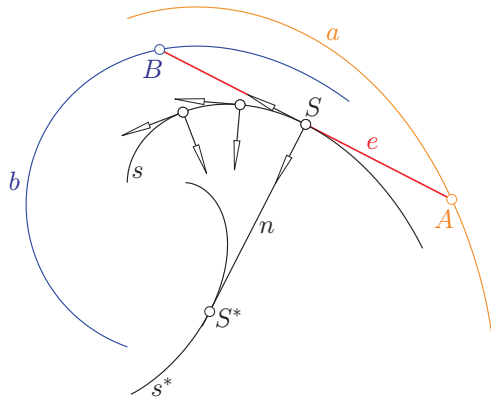


planar case B

$\Sigma/\Sigma^*$  ... motion of a straight line  $n$  rolling on a curve  $s^*$

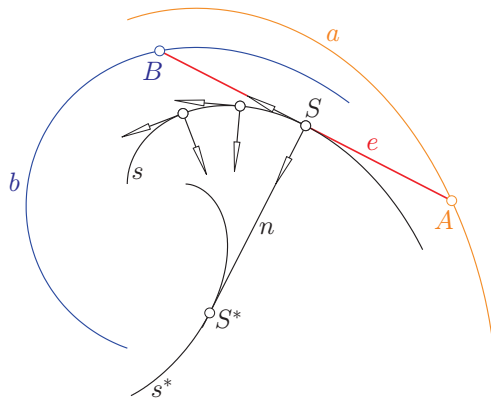
$S$  ... midpoint of  $AB$

$\Sigma/\Sigma^*$  ... Frenet motion along the path  $s$  of  $S$



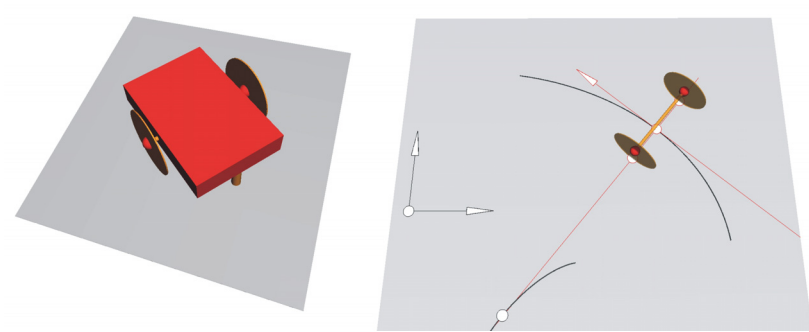
planar case B

$s$  is a tractrix with respect to  $a$  and  $b$



Rope Ladder Problem: The Planar Case

mobile robot with two wheels



## Rope Ladder Problem: The Spatial Case

Rope Ladder Problem: The Spatial Case

ruled surface  $\Phi$  generated by the motion of  $e = AB$ :

$$\mathbf{y}(t, u) = \mathbf{x}(t) + u\mathbf{e}(t) \text{ with } \langle \mathbf{e}, \mathbf{e} \rangle = 1$$



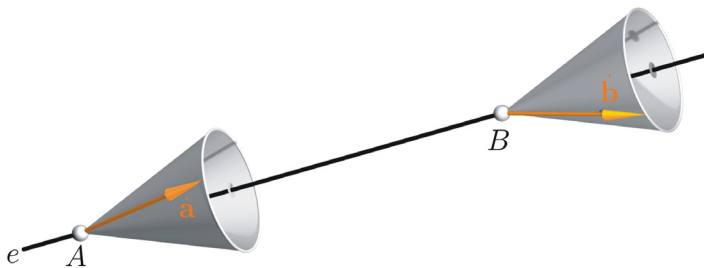
Rope Ladder Problem: The Spatial Case

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$$\mathbf{y}(t, u) = \mathbf{x}(t) + u\mathbf{e}(t) \text{ with } \langle \mathbf{e}, \mathbf{e} \rangle = 1$$

$$A \dots \mathbf{a}(t) = \mathbf{y}(t, a)$$

$$B \dots \mathbf{b}(t) = \mathbf{y}(t, a + d)$$







Rope Ladder Problem: The Spatial Case

$$(2a + d)\langle \dot{\mathbf{e}}, \dot{\mathbf{e}} \rangle = -2\langle \dot{\mathbf{x}}, \dot{\mathbf{e}} \rangle$$

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$$\dot{\mathbf{e}} = 0 \iff \mathbf{e} = \mathbf{c}$$

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$\Phi$  is a cylinder (trivial case)

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$$\dot{\mathbf{x}} = 0 \iff \mathbf{x} = \mathbf{c} = \mathbf{s}$$

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$\Phi$  is a cone and  $A, B$  are symmetric w.r.t. its vertex  $S$  (trivial case)

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spatial case C

$$\dot{\mathbf{e}}, \dot{\mathbf{x}} \neq 0$$

Rope Ladder Problem: The Spatial Case

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$$\dot{\mathbf{e}}, \dot{\mathbf{x}} \neq 0$$

$$a + \frac{d}{2} = -\frac{\langle \dot{\mathbf{x}}, \dot{\mathbf{e}} \rangle}{\langle \dot{\mathbf{e}}, \dot{\mathbf{e}} \rangle}$$

Rope Ladder Problem: The Spatial Case

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The midpoint  $S$  of  $AB$  is the striction (cuspidal) point on  $e = AB$ .



**Rope Ladder Problem:** The Spatial Case

An Example: screw motion

An Example: screw motion

all points on a common right cylinder around the screw axis have paths of equal length

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all points on a common right cylinder around the screw axis have paths of equal length

striction curve  $s$  is the helix generated by  $S$

## Rope Ladders: An interpolation problem

**Rope Ladders:** An Interpolation Problem

**Given:** series of positions  $A_iB_i$ ,  $i = 1, \dots, n$  of the rod  $AB$

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**Wanted:** motion  $\Sigma/\Sigma^*$  which

- a) moves  $AB$  through the given positions and
- b) guarantees equal path lengths of  $A$  and  $B$

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*Step 1:* Find a suitable curve  $s$  interpolating the midpoints  $S_i$  of  $A_iB_i$



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**Construction:**

*Step 1:* Find a suitable curve  $s$  interpolating the midpoints  $S_i$  of  $A_iB_i$

*Step 2:* Find a ruled surface  $\Phi$  that interpolates  $e_i = A_iB_i$  and whose striction curve is  $s$

**Rope Ladders:** An Interpolation Problem

*Step 1:* curve  $s \dots s(t)$  with  $s(t_i) = \frac{\mathbf{a}_i + \mathbf{b}_i}{2} =: \mathbf{s}_i, i = 1, \dots, n$

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$$\langle \mathbf{s}', \mathbf{s}' \rangle = 1;$$

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$$\langle \mathbf{s}', \mathbf{e}' \rangle = 0 \quad (\text{S})$$

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$$\langle \mathbf{s}', \mathbf{e}' \rangle = 0 \quad (\text{S})$$

$$\langle \mathbf{e}, \mathbf{e} \rangle = 1 \quad (\text{U})$$

**Rope Ladders:** An Interpolation Problem

$\sigma := \angle(\mathbf{e}, \mathbf{s}') \dots$  striction of  $\Phi$



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$\mathbf{t} = \mathbf{s}'$ ,  $\mathbf{h} = \frac{1}{|\mathbf{s}''|} \mathbf{s}''$ ,  $\mathbf{b} = \mathbf{t} \times \mathbf{p} \dots$  Frenet frame of  $\mathbf{s}$

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$$\varepsilon \quad \dots \quad \langle \mathbf{s}', \mathbf{e} \rangle \quad - \quad \cos \sigma \quad = \quad 0$$

$$\varepsilon_1 \quad \dots \quad \langle \mathbf{s}'', \mathbf{e} \rangle \quad + \quad \sigma' \sin \sigma \quad = \quad 0$$

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$$\varepsilon \dots \langle \mathbf{s}', \mathbf{e} \rangle - \cos \sigma = 0$$

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developable surface  $\Gamma$  with generators  $g$  parallel to  $\mathbf{b}$

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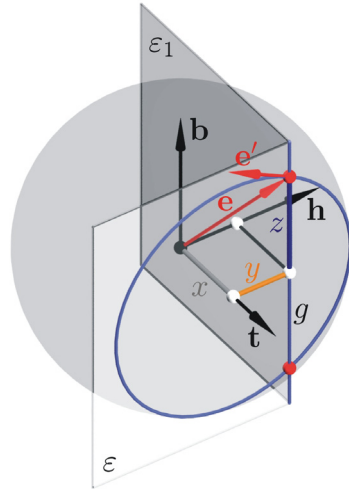
developable surface  $\Gamma$  with generators  $g$  parallel to  $\mathbf{b}$

$$\varepsilon_1 \dots \langle \mathbf{h}, \mathbf{e} \rangle + \frac{\sigma' \sin \sigma}{\kappa} = 0$$

The intersection of  $\Gamma$  with the unit sphere contains the spherical generator image  $\mathbf{e} = \mathbf{e}(s)$  of  $\Phi$

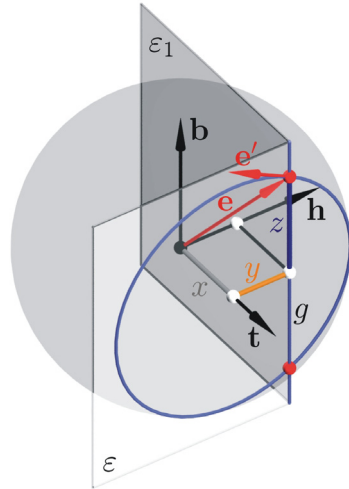


Rope Ladders: An Interpolation Problem



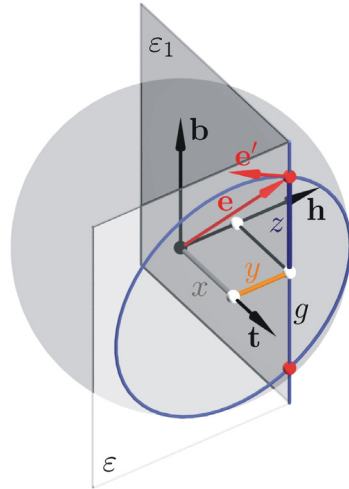
Rope Ladders: An Interpolation Problem

$$\begin{aligned}x &= \cos \sigma \\y &= -\frac{\sigma' \sin \sigma}{\kappa} \\z &= \pm \sin \sigma \sqrt{1 - \frac{\sigma'^2}{\kappa^2}}\end{aligned}$$



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$$\begin{aligned}x &= \cos \sigma \\y &= -\frac{\sigma' \sin \sigma}{\kappa} \\z &= \pm \sin \sigma \sqrt{1 - \frac{\sigma'^2}{\kappa^2}}\end{aligned}$$



$$\mathbf{e} = \cos \sigma \cdot \mathbf{t} - \sigma' \sin \sigma \cdot \mathbf{h} \pm \sin \sigma \sqrt{1 - \frac{\sigma'^2}{\kappa^2}} \cdot \mathbf{b}$$

**Rope Ladders:** An Interpolation Problem

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Special Cases

$$\mathbf{e} = \cos \sigma \cdot \mathbf{t} - \sigma' \sin \sigma \cdot \mathbf{h} \pm \sin \sigma \sqrt{1 - \frac{\sigma'^2}{\kappa^2}} \cdot \mathbf{b}$$

### Special Cases

- $\sigma' = 0$ :  $\Phi$  is a ruled surface of constant striction;  
 $\mathbf{e} \in [\mathbf{t}, \mathbf{b}]$

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- $\kappa = 0$  (striction curve  $s$  is a straight line):  
a solution is possible only if  $\sigma' = 0$ , i.e.;  $\sigma = \text{const.}$   
 $\Phi$  is a ruled surface of constant slope and a straight line as striction curve.



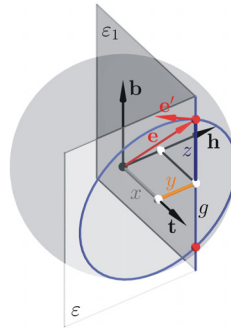
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$$\begin{aligned} \varepsilon \quad \dots \quad \langle \mathbf{t}, \mathbf{e} \rangle - \cos \sigma &= 0 \\ \varepsilon_1 \quad \dots \quad \langle \mathbf{h}, \mathbf{e} \rangle + \frac{\sigma' \sin \sigma}{\kappa} &= 0 \end{aligned}$$

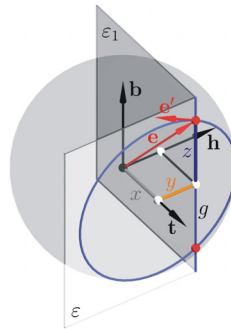




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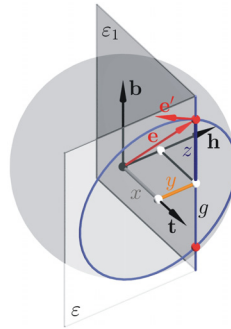
Construct a striction function  $\sigma$  with

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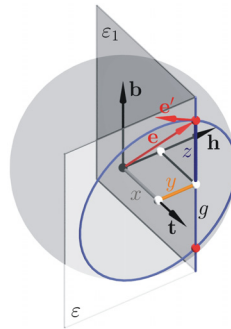
Construct a striction function  $\sigma$  with

$$\left. \begin{aligned} \sigma'(\tau)^2 &\leq \kappa^2(\tau) \\ \sigma(\tau_i) &= \arccos(\mathbf{s}'(\tau_i), \mathbf{e}_i) \\ \sigma'(\tau_i) &= -\frac{\langle \mathbf{s}''(\tau_i), \mathbf{e}_i \rangle}{\sin \sigma(\tau_i)} \end{aligned} \right\}, \quad i = 1, \dots, n$$

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Construct  $\sigma$  as a Hermite interpolant.

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